

Stress Analysis of Bullet-Holes on the Boeing 737 Fuselage



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ES 240 Project

Overview

- Introduction / Objectives
- Variable / Parameter Definitions
- Case 1: Single Bullet Hole
- Case 2: Two Bullet Holes
- Case 3: Single Bullet Hole Near Window
- Case 4: Two Bullet Holes Near Window
- Case 5: Two Holes Approaching Each Other
- Extra: Fracture due to Crack-Propagation

Introduction / Objectives

- Introduction:

- How we came about doing this project
- Why this analysis is important
- How is this applicable to real-life engineering

- Objectives

- Go through various case examples
- Use ABAQUS CAE to model and view stress, analyze more complex scenarios
- Compare results with theoretical calculations

Variable / Parameter Definitions

- Material Constants:

- Aluminum 7079-T6*

- Young's Modulus:

- $E = 71.7 \text{ GPa}$

- Poisson's Ratio:

- $\nu = 0.33$

- Density: 2740 kg/m^3

- Physical Dimensions

- Dimension of Fuselage:

- Radius = 5 m

- Thickness = 0.2 m

- Dimension of Bullet Hole

- Diameter = 1

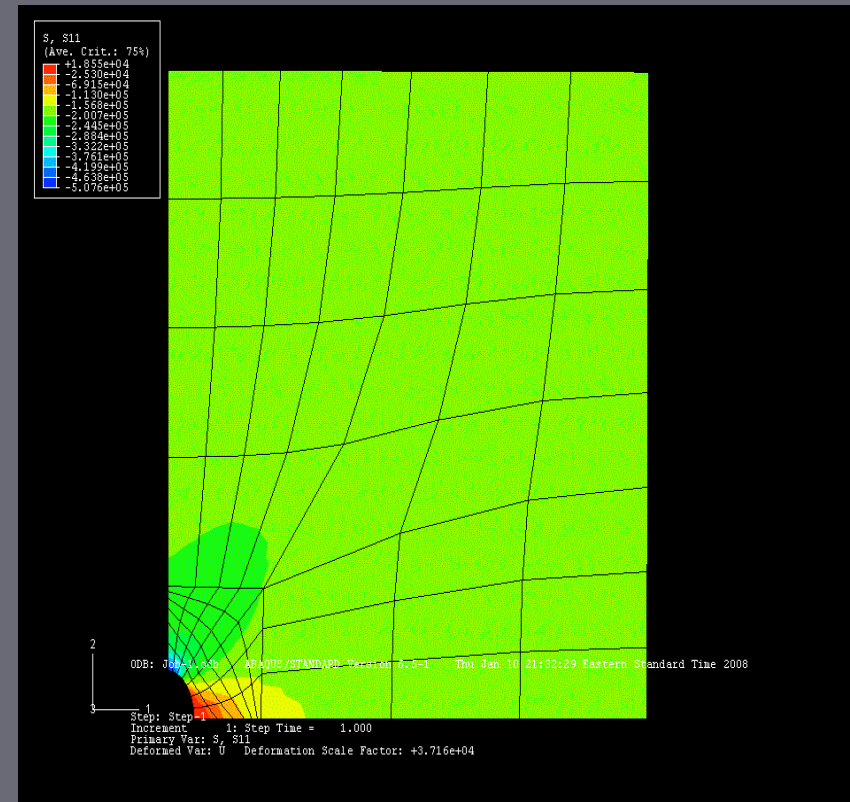
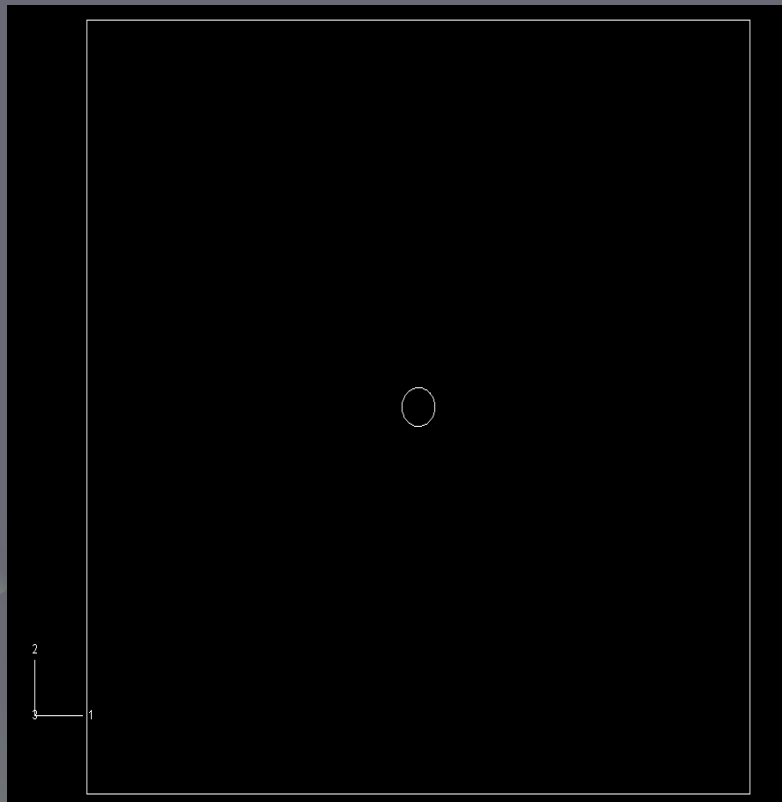
- Dimension of Window

- 20×10

- General Assumptions and Simple Calculations:

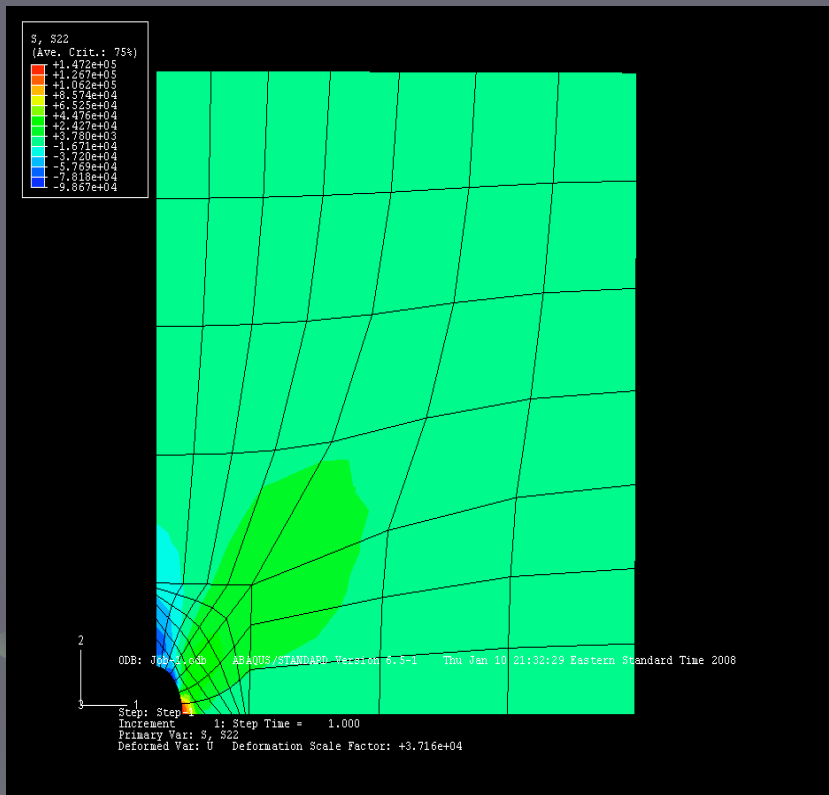
- Uniaxial Load in Compression: 0.19 MPa*

Case 1: Single Bullet Hole

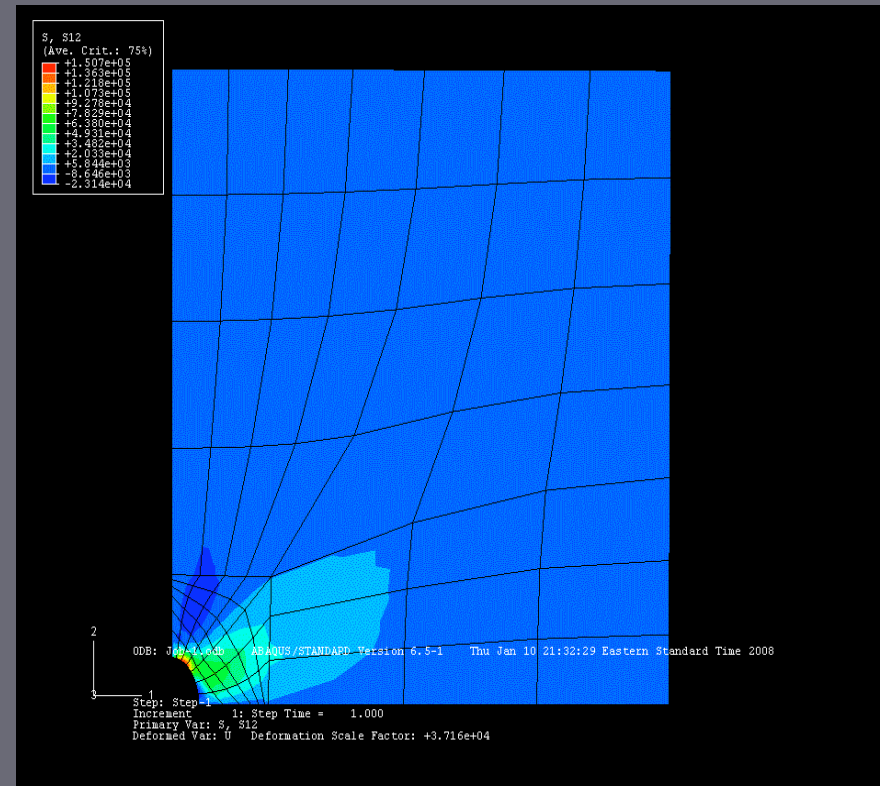


● S11

Case 1: ABAQUS CAE Analysis

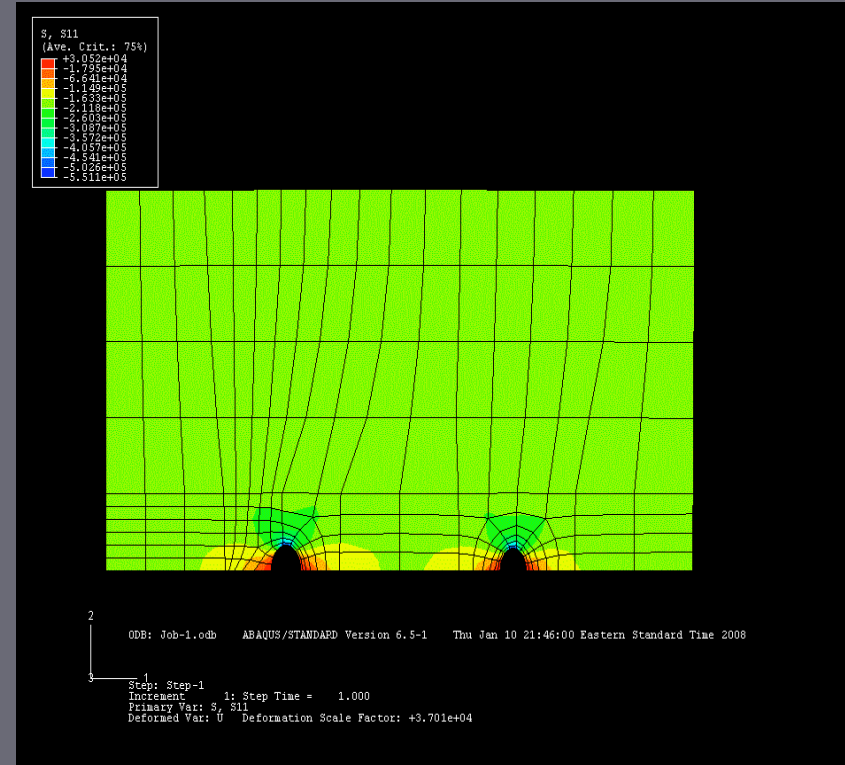
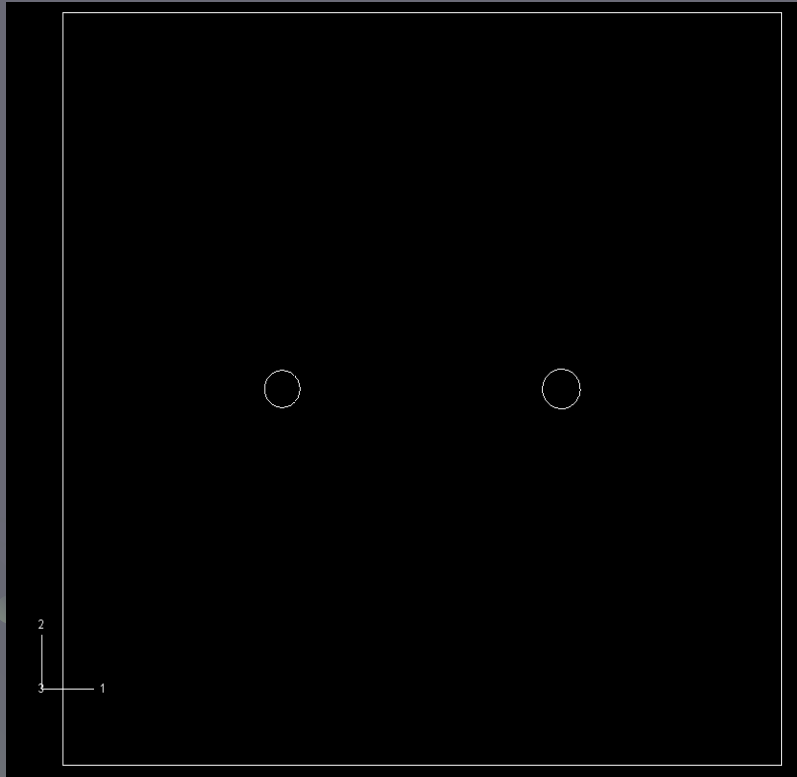


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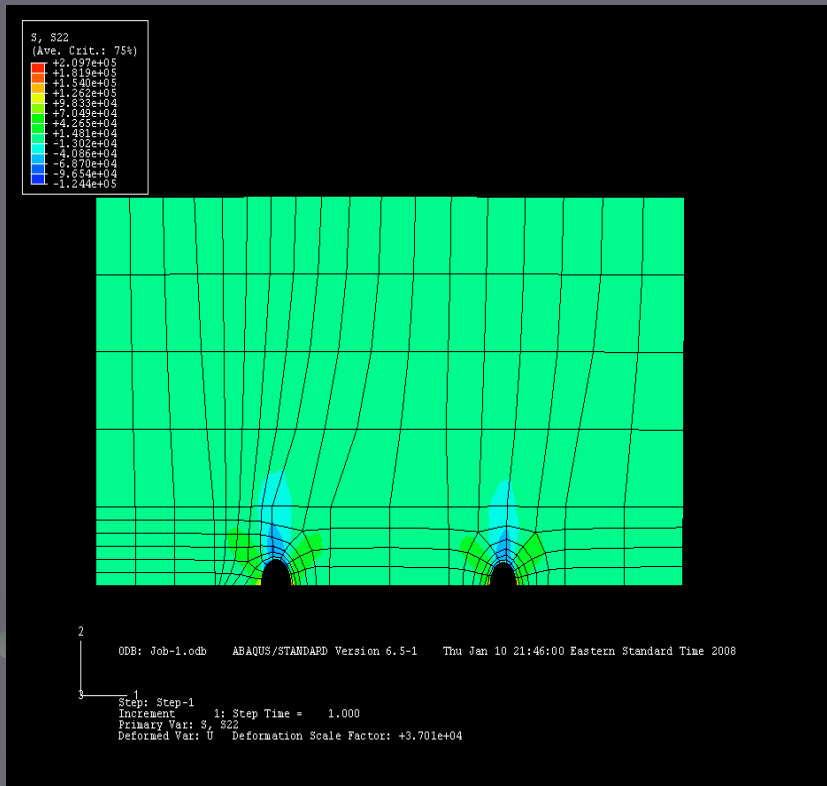
● S12

Case 2: Two Bullet Holes

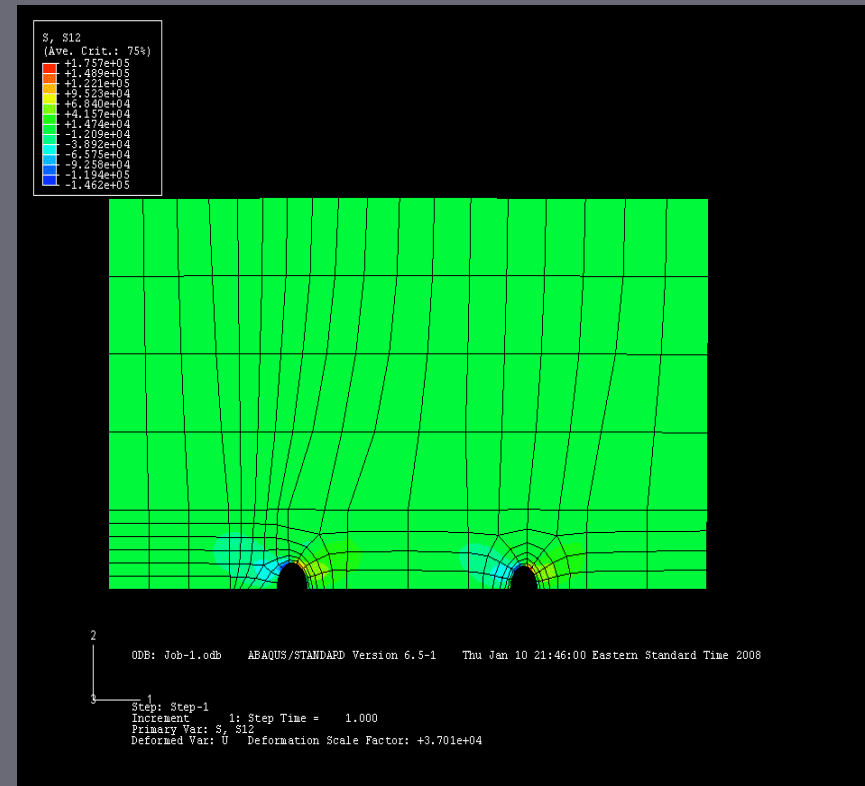


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Case 2: ABAQUS CAE Analysis

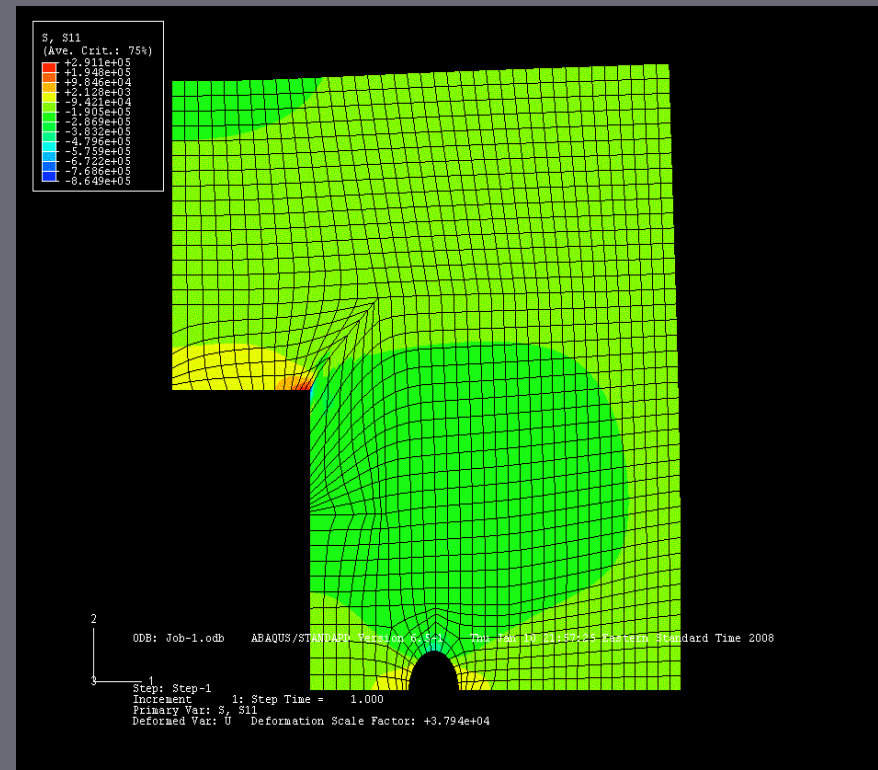
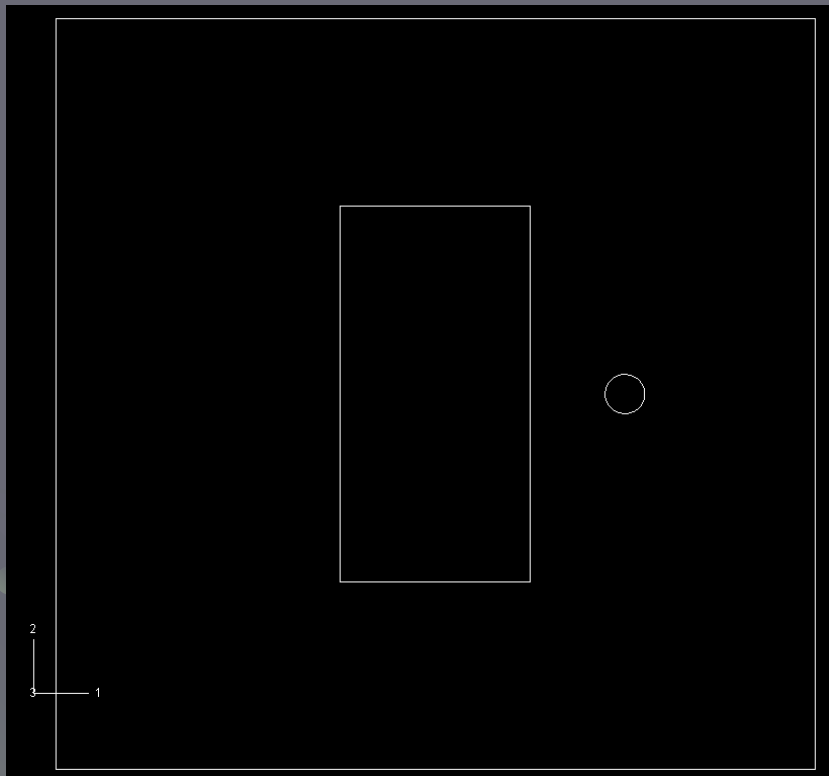


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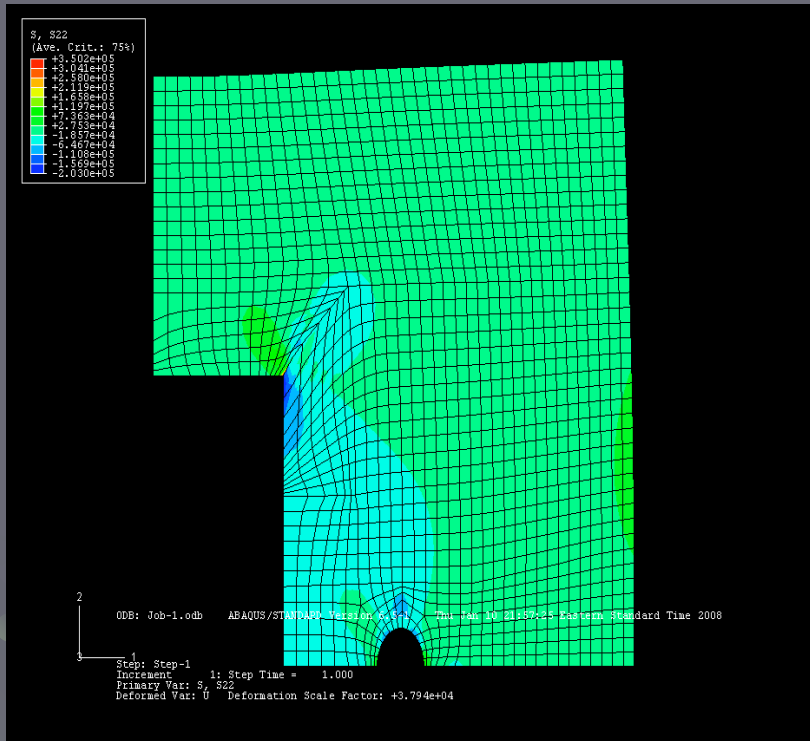
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Case 3: Single Bullet Hole Near Window

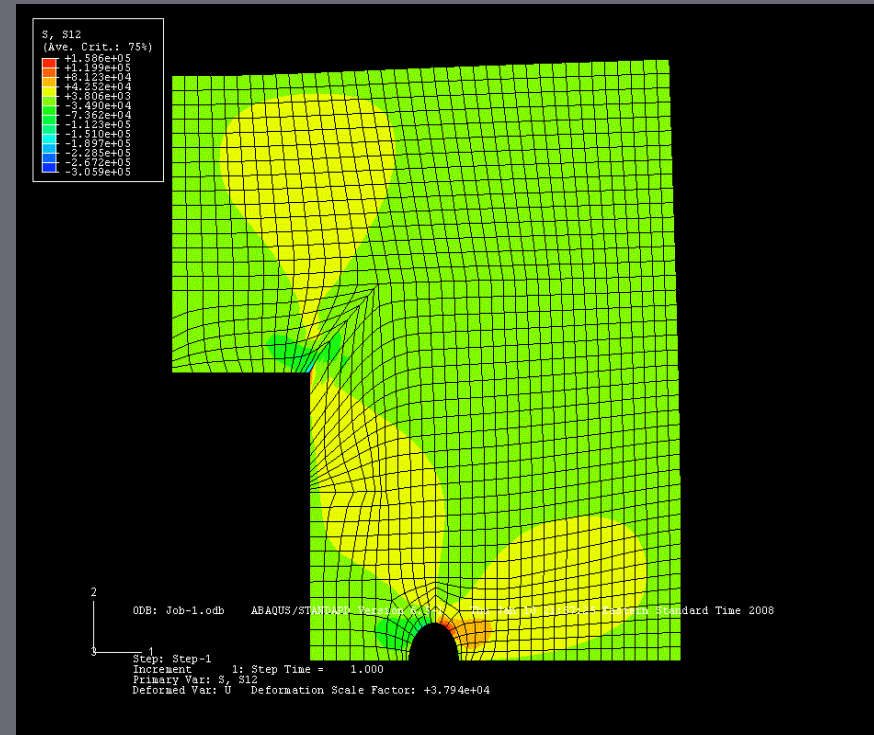


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Case 3: ABAQUS CAE Analysis

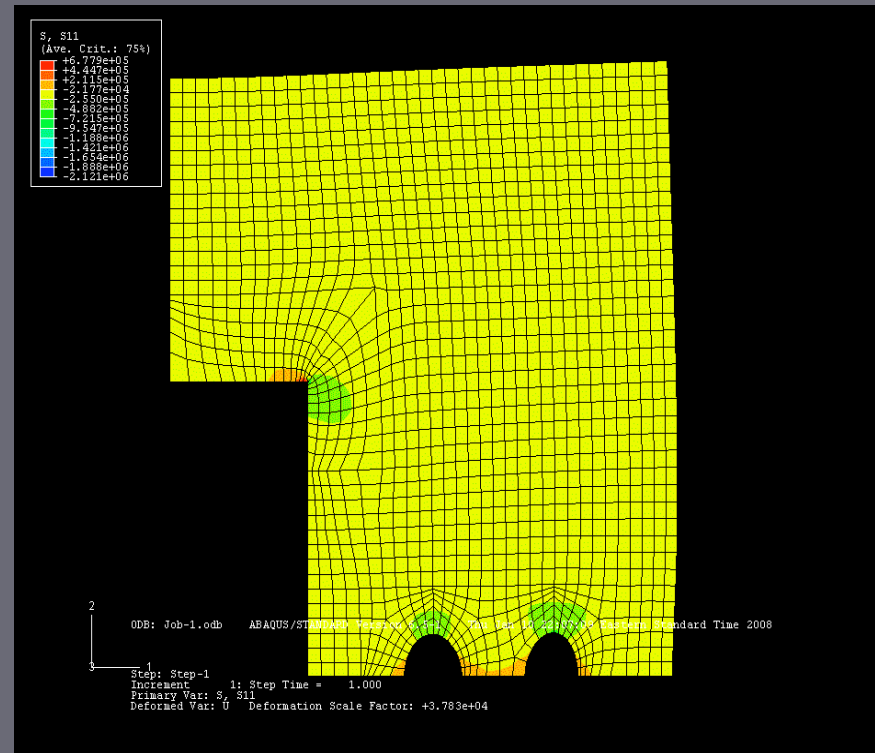
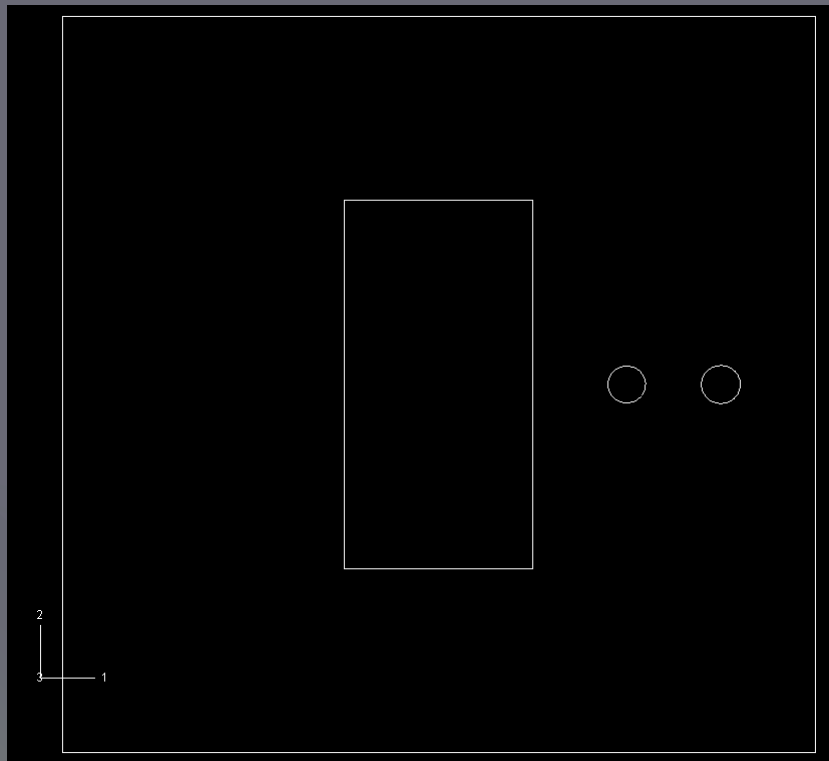


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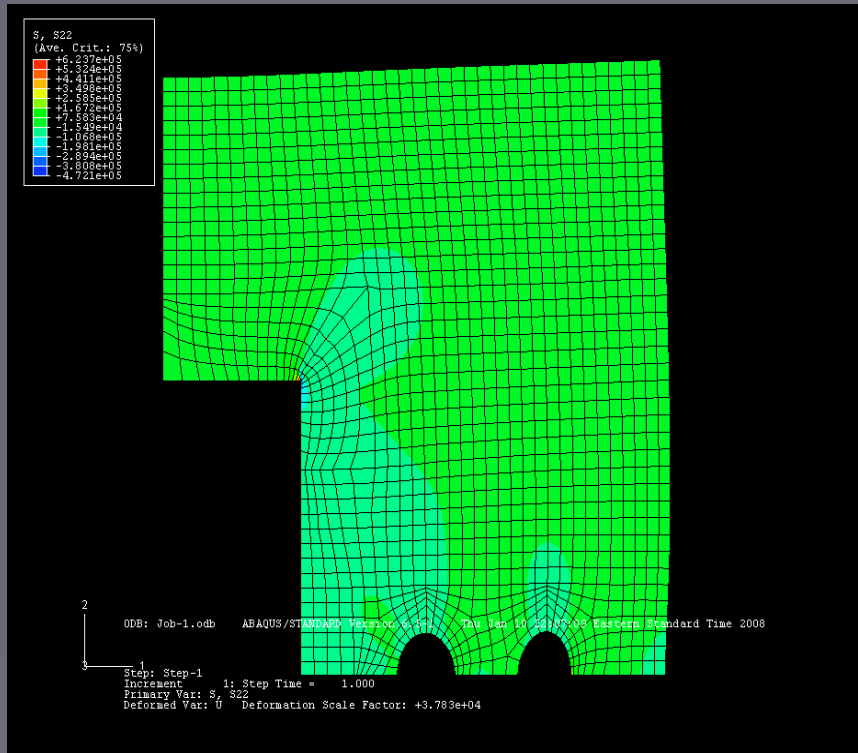
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Case 4: Two Bullet Holes Near Window

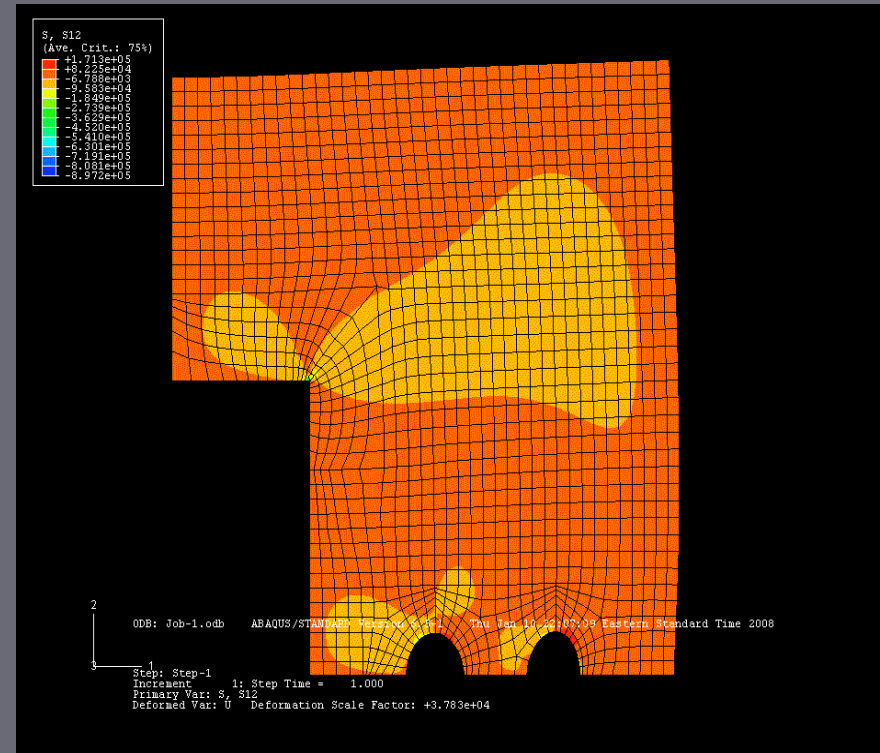


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Case 4: ABAQUS CAE Analysis

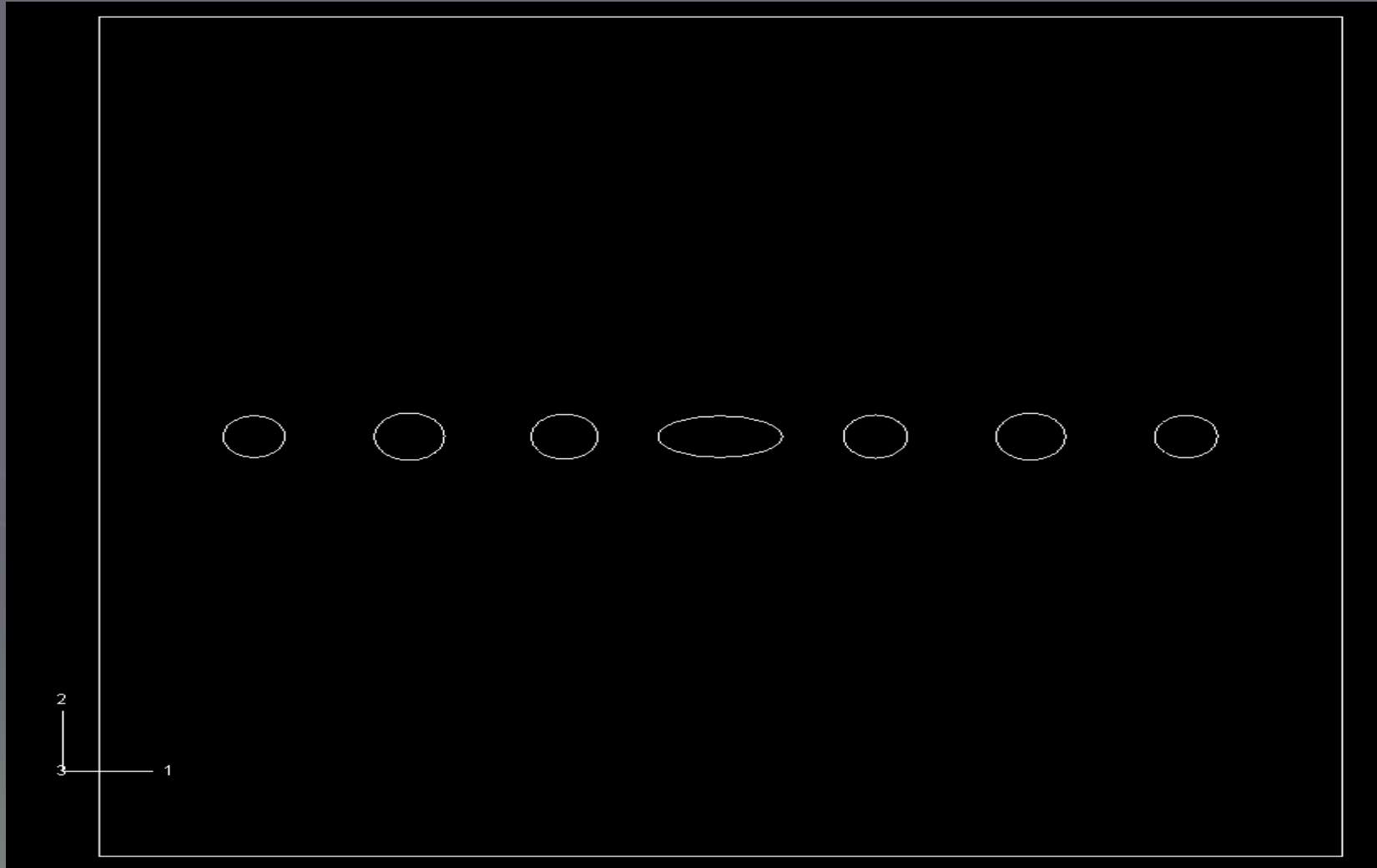


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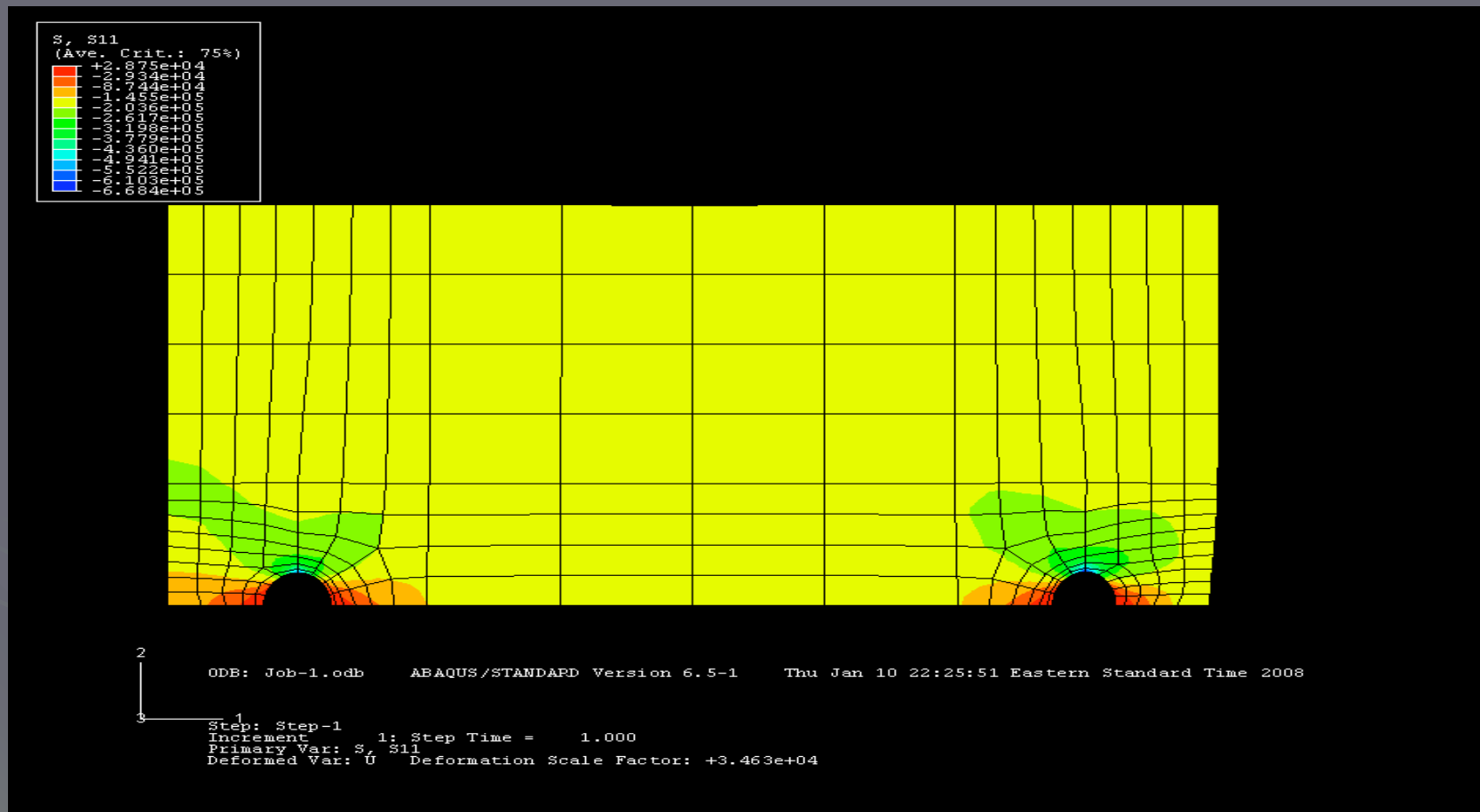


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Case 5: Two Holes Approaching Each Other

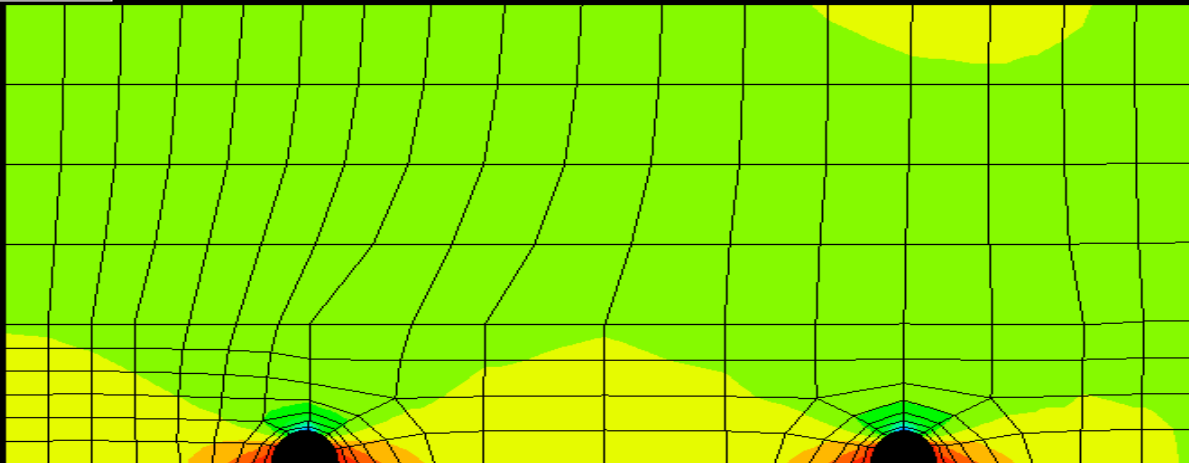
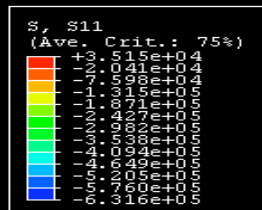


Case 5: ABAQUS CAE Analysis



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Case 5: ABAQUS CAE Analysis



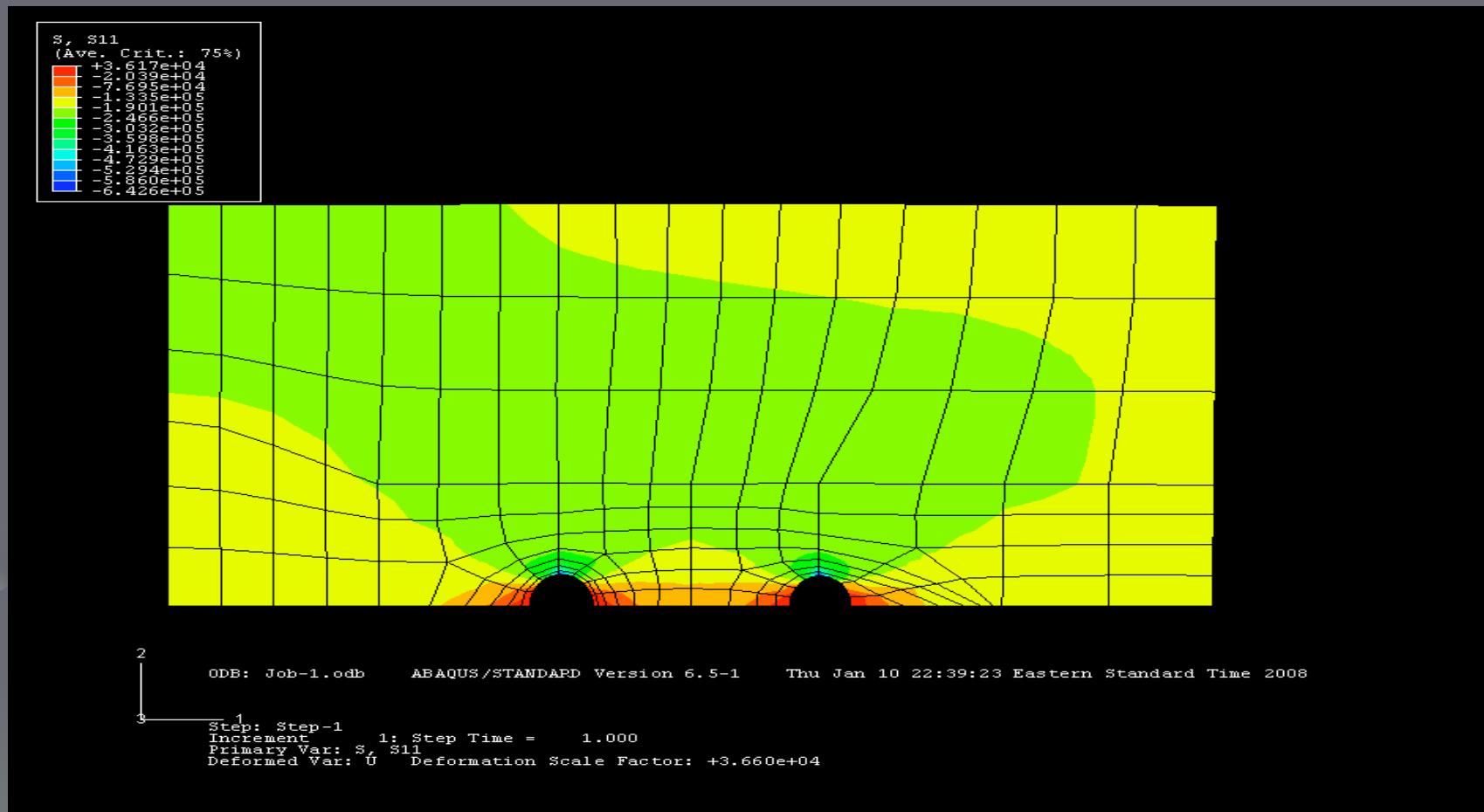
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Step: Step-1
Increment 1: Step Time = 1.000
Primary Var: S, S11
Deformed Var: U Deformation Scale Factor: +3.646e+04

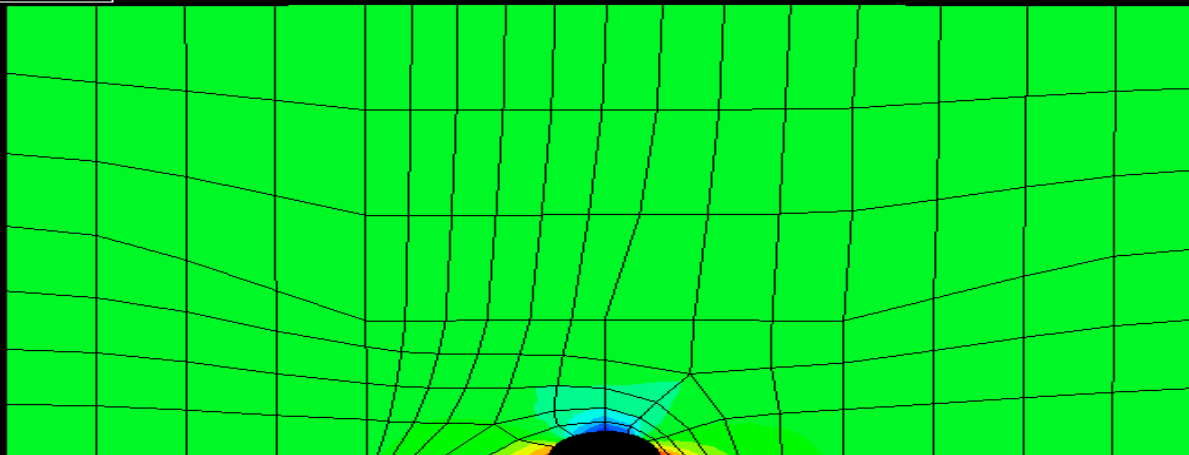
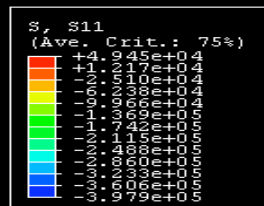
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Case 5: ABAQUS CAE Analysis



● S11

Case 5: ABAQUS CAE Analysis



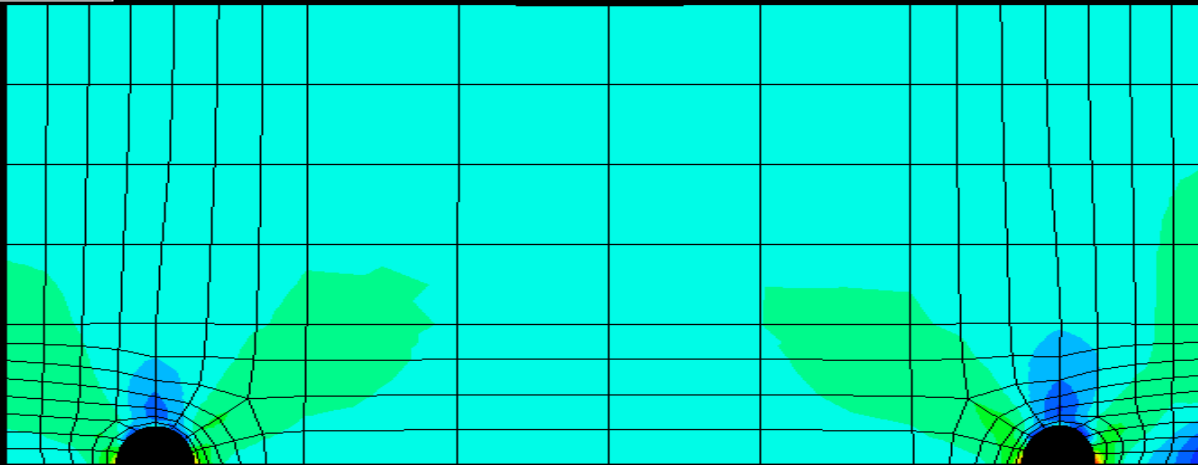
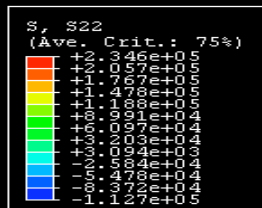
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● S11

Case 5: ABAQUS CAE Analysis



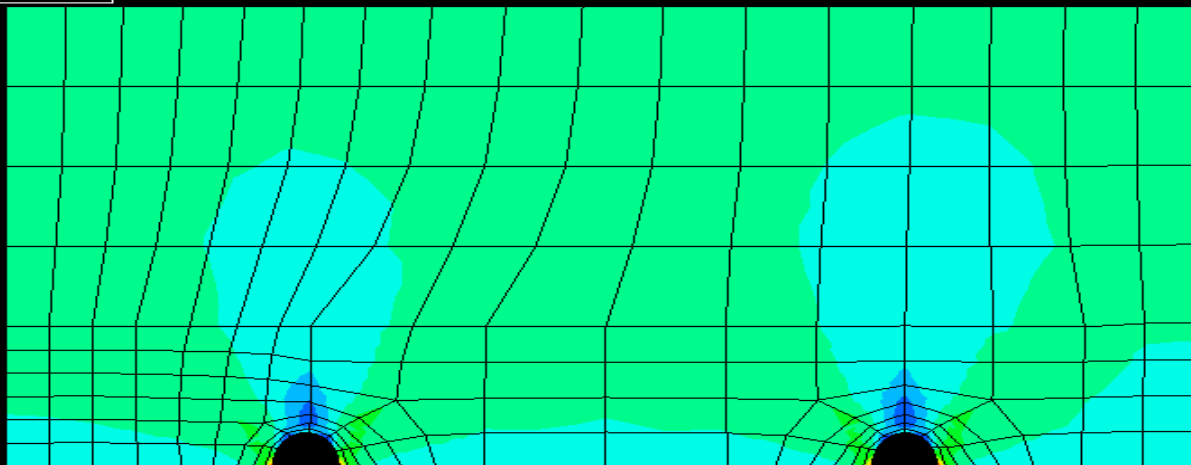
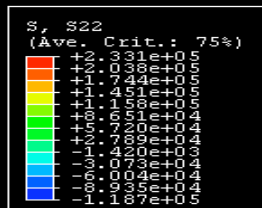
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Step: Step-1
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Primary Var: S, S22
Deformed Var: U Deformation Scale Factor: +3.463e+04

● S22

Case 5: ABAQUS CAE Analysis



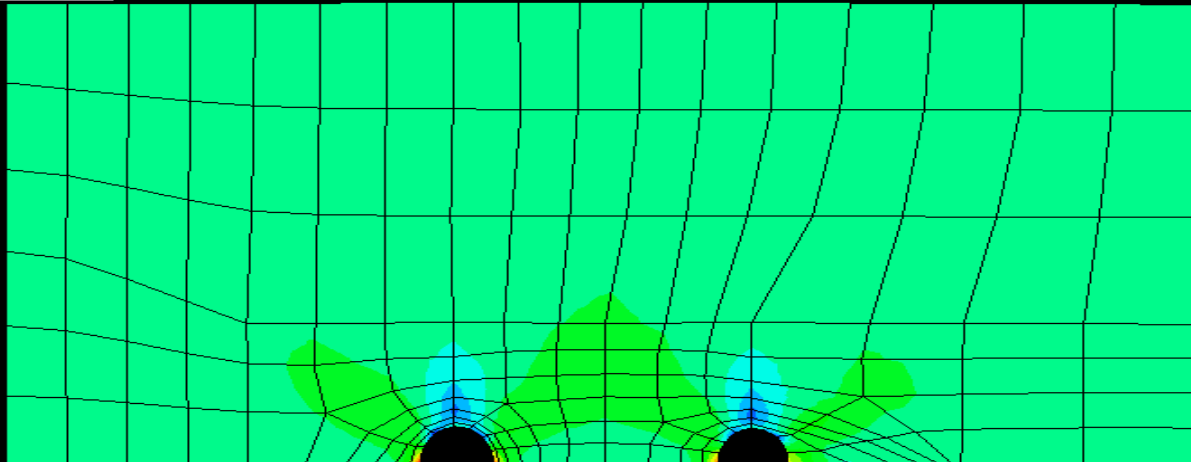
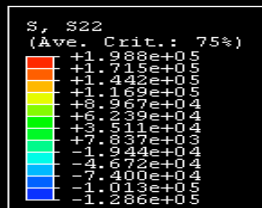
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Deformed Var: U Deformation Scale Factor: +3.646e+04

● S22

Case 5: ABAQUS CAE Analysis



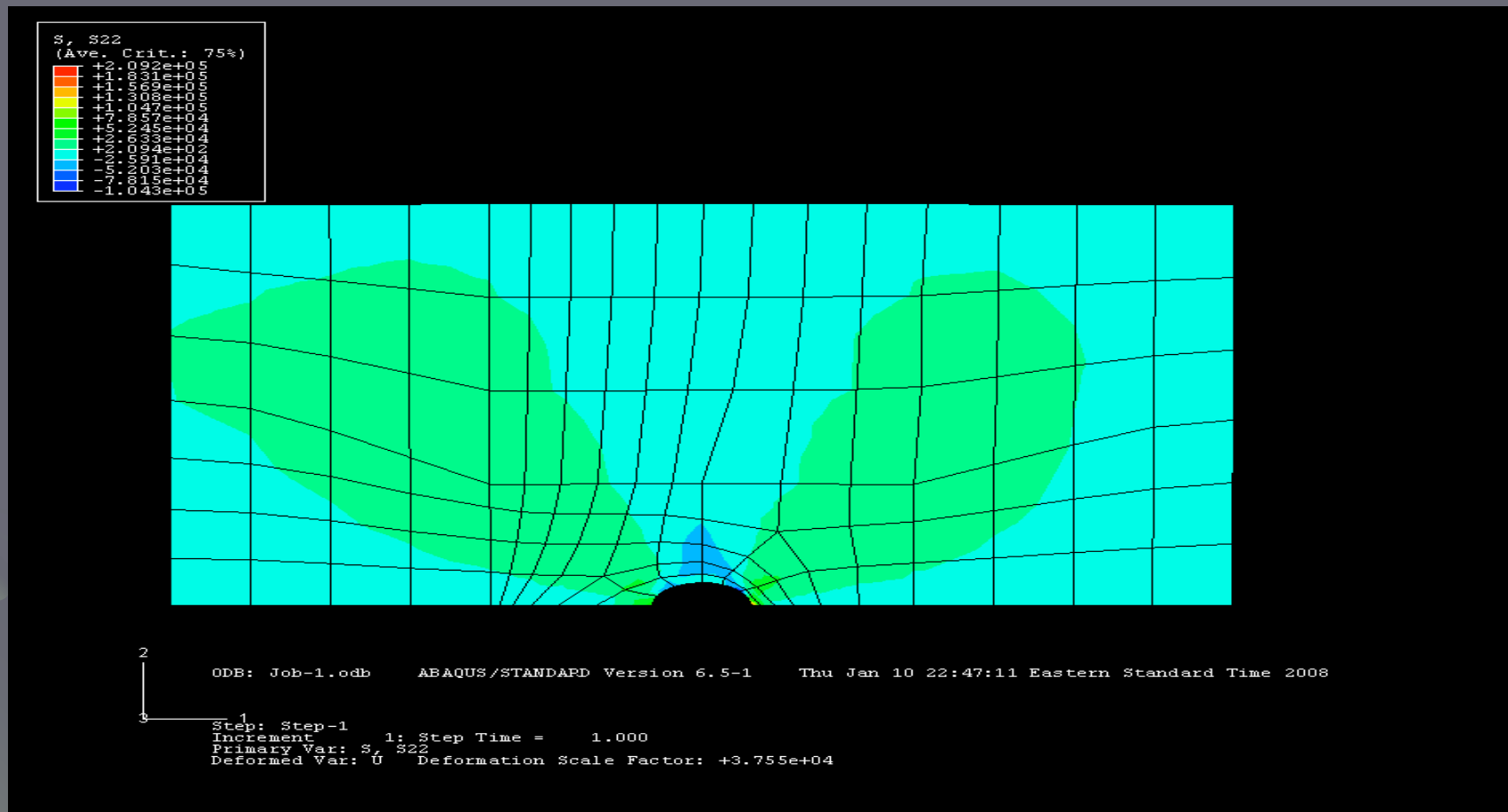
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Deformed Var: U Deformation Scale Factor: +3.660e+04

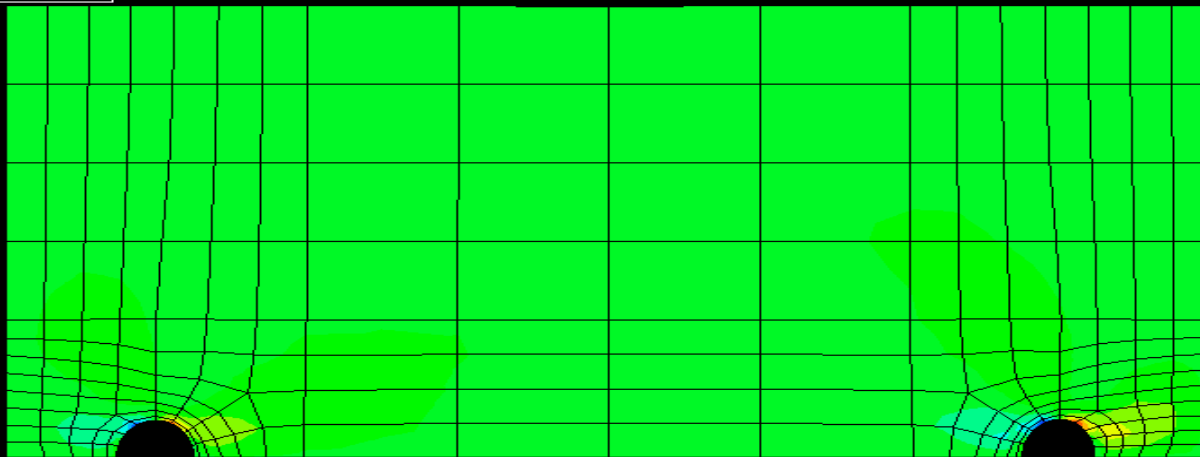
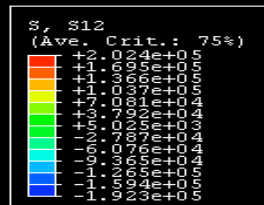
● S22

Case 5: ABAQUS CAE Analysis



● S22

Case 5: ABAQUS CAE Analysis



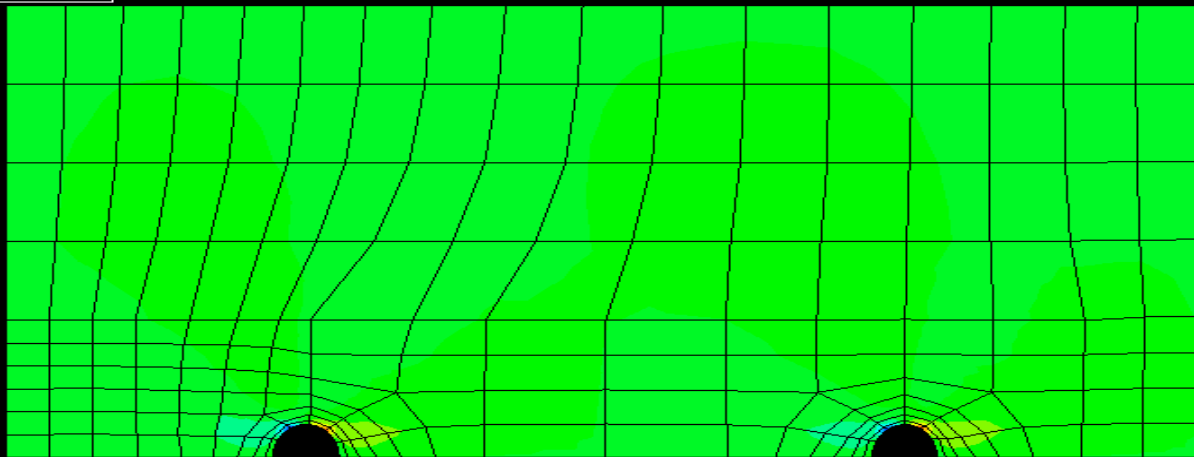
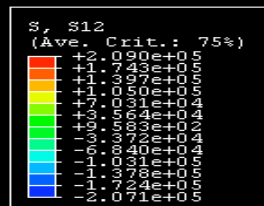
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3 Step: Step-1
  Increment: 1; Step Time = 1.000
  Primary Var: S11
  Deformed Var: U1 Deformation Scale Factor: +3.463e+04

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● S12

Case 5: ABAQUS CAE Analysis



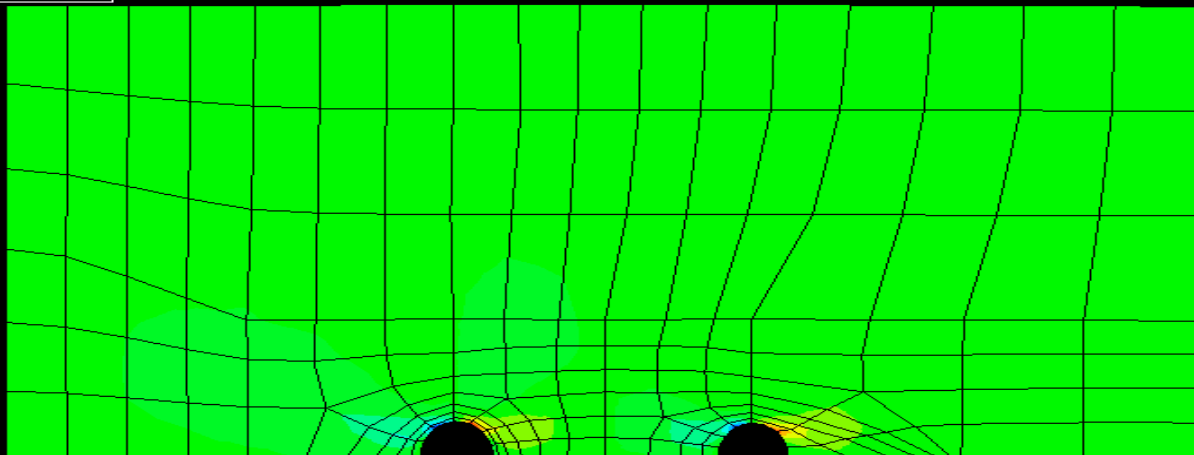
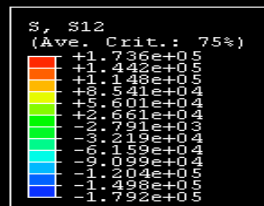
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● S12

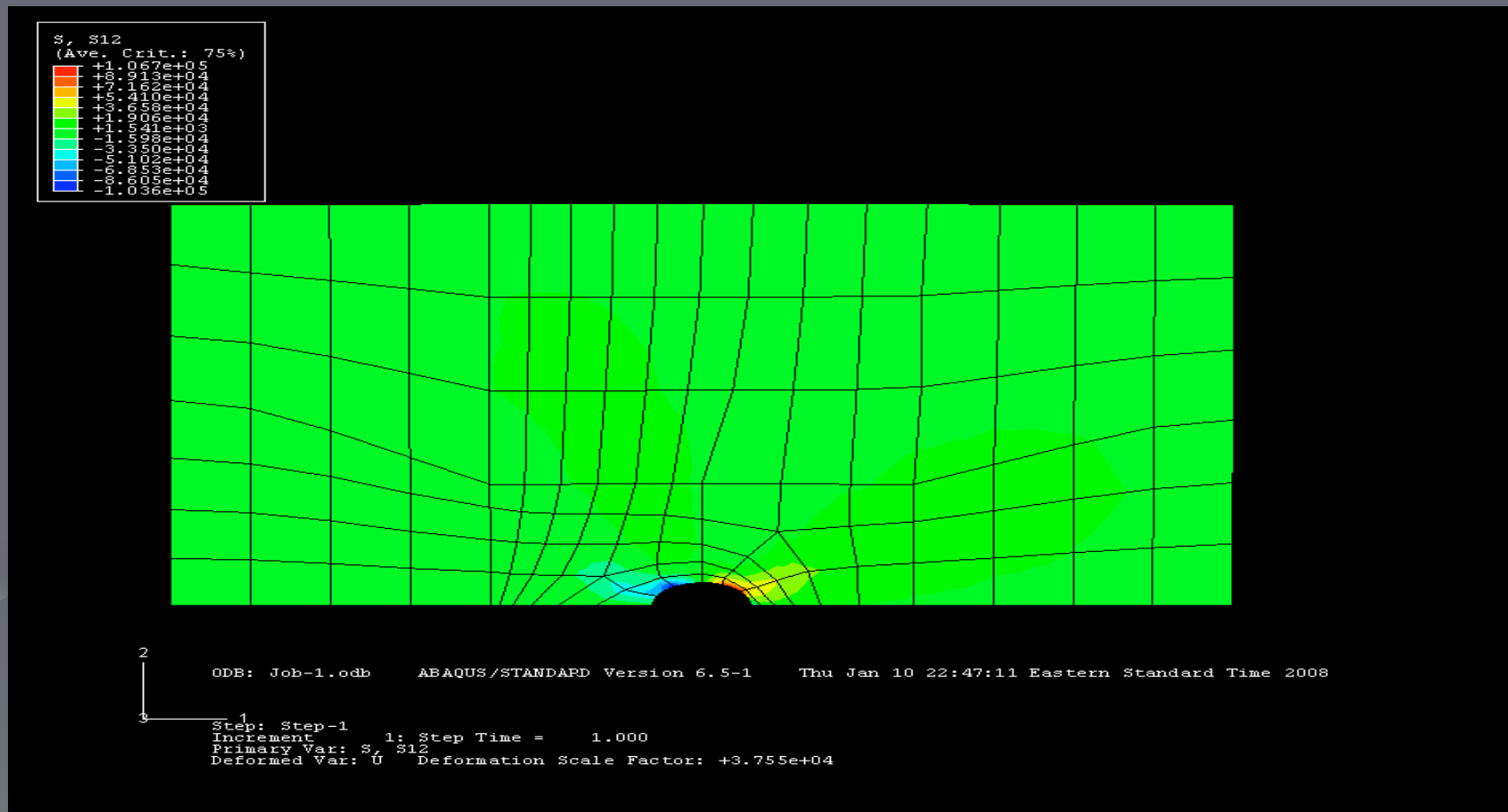
Case 5: ABAQUS CAE Analysis



2
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ODB: Job-1.odb ABAQUS/STANDARD Version 6.5-1 Thu Jan 10 22:39:23 Eastern Standard Time 2008
Step: Step-1
Increment 1: Step Time = 1.000
Primary Var: S, S12
Deformed Var: U Deformation Scale Factor: +3.660e+04

● S12

Case 5: ABAQUS CAE Analysis



● S12

Orthotropic Stress Equations (Ellipse)

Stress-Strain Relations:

$$\varepsilon_{XX} = \frac{1}{E_X} \sigma_{XX} - \frac{\nu_{YX}}{E_Y} \sigma_{YY}$$

$$\varepsilon_{YY} = \frac{1}{E_Y} \sigma_{YY} - \frac{\nu_{XY}}{E_X} \sigma_{XX}$$

$$\varepsilon_{XY} = \frac{1}{\mu_{XY}} \sigma_{XY}$$

Compatibility Equation:

$$\frac{\partial^2 \varepsilon_{XX}}{\partial y^2} + \frac{\partial^2 \varepsilon_{YY}}{\partial x^2} = \frac{\partial^2 \varepsilon_{XY}}{\partial x \partial y}$$

Airy Stress Function:

$$\sigma_{XX} = \frac{\partial^2 F}{\partial y^2} \quad \sigma_{YY} = \frac{\partial^2 F}{\partial x^2} \quad \sigma_{XY} = -\frac{\partial^2 F}{\partial x \partial y}$$

Differential Equation 1:

$$\frac{1}{E_Y} \frac{\partial^4 F}{\partial x^4} + \left(\frac{1}{\mu_{XY}} - 2 \frac{\nu_{XY}}{E_X} \right) \frac{\partial^4 F}{\partial x^2 \partial y^2} + \frac{1}{E_X} \frac{\partial^4 F}{\partial y^4} = 0$$

Differential Equation Simplified:

$$\frac{\partial^4 F}{\partial x^4} + 2\kappa \frac{\partial^4 F}{\partial x^2 \partial \eta^2} + \frac{\partial^4 F}{\partial \eta^4} = 0$$

where:

$$\kappa = \frac{\sqrt{E_X E_Y}}{2} \left(\frac{1}{\mu_{XY}} - \frac{2\nu_{XY}}{E_X} \right)$$

$$\eta = \xi y \quad \text{and} \quad \xi = \sqrt[4]{\frac{E_X}{E_Y}}$$

Orthotropic Stress Equations (Ellipse)

Solve for Roots of Equation:

$$r_1 = ie^{\frac{\phi}{2}} \quad r_2 = ie^{-\frac{\phi}{2}}$$

$$r_3 = -ie^{\frac{\phi}{2}} \quad r_4 = -ie^{-\frac{\phi}{2}}$$

Specific Solution Form:

$$F = \text{Re} \left\{ \frac{A}{2y_1^2} \left[\frac{1}{2} (z_1 - w_1)^2 + y_1^2 \log(z_1 + w_1) \right] + \frac{B}{2y_2^2} \left[\frac{1}{2} (z_2 - w_2)^2 + y_2^2 \log(z_2 + w_2) \right] \right\} + \frac{P\eta^2}{2\xi^2}$$

Biharmonic Equation Form:

$$\left(\frac{\partial^2}{\partial x^2} + \alpha^2 \frac{\partial^2}{\partial \eta^2} \right) \left(\frac{\partial^2}{\partial x^2} + \beta^2 \frac{\partial^2}{\partial \eta^2} \right) F = 0$$

where: $\alpha^2 = e^{\phi}$
 $\beta^2 = e^{-\phi}$

where: $z_1 = x + i\alpha\eta$

$$z_2 = x + i\beta\eta$$

$$w_1 = \sqrt{z_1^2 - y_1^2}$$

$$w_2 = \sqrt{z_2^2 - y_2^2}$$

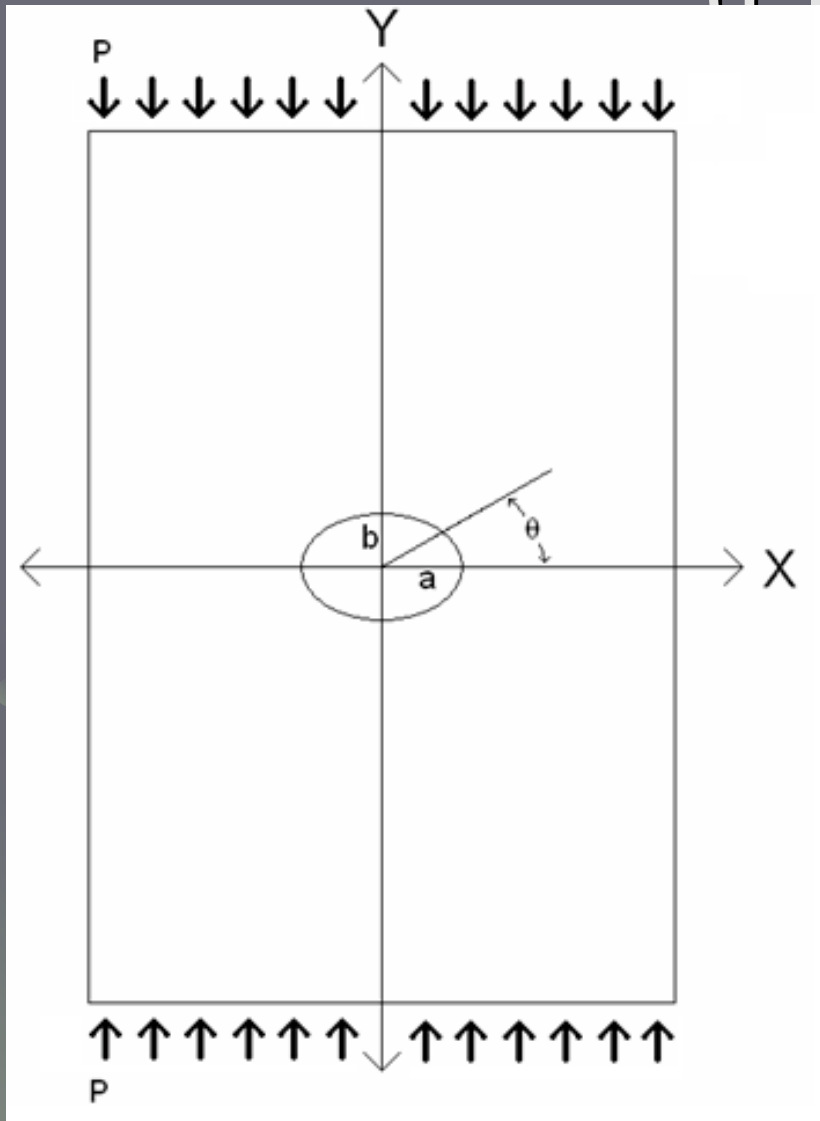
$$y_1^2 = a^2 - \alpha^2 \xi^2 b^2$$

$$y_2^2 = a^2 - \beta^2 \xi^2 b^2$$

General Solution Form:

$$F = \text{Re} [F_1(x + i\alpha\eta) + F_2(x + i\beta\eta)] + F_P(x, \eta)$$

Orthotropic Stress Equations (Ellipse)



$$x = a \cos \theta$$

$$\eta = \xi y = \xi b \sin \theta$$

$P = \text{pressure}$

Use Boundary Conditions to solve for constants of integration and unknown variables...

Orthotropic Stress Equations (Ellipse)

$$\frac{\partial^2 F}{\partial x^2} = \sigma_{YY} = \text{Re} \left\{ \frac{Pb}{\xi(\alpha - \beta)} \left[\frac{(a + \alpha\xi b)}{w_1(z_1 + w_1)} - \frac{(a + \beta\xi b)}{w_2(z_2 + w_2)} \right] \right\}$$

$$\frac{\partial^2 F}{\partial y^2} = \sigma_{XX} = \text{Re} \left\{ \frac{P\xi b}{(\alpha - \beta)} \left[\frac{-\alpha^2(a + \alpha\xi b)}{w_1(z_1 + w_1)} + \frac{\beta^2(a + \beta\xi b)}{w_2(z_2 + w_2)} \right] \right\} + P$$

$$\frac{\partial^2 F}{\partial x \partial \eta} = \sigma_{XY} = \text{Re} \left\{ \frac{Pb}{(\alpha - \beta)} \left[\frac{-\alpha i(a + \alpha\xi b)}{w_1(z_1 + w_1)} + \frac{\beta i(a + \beta\xi b)}{w_2(z_2 + w_2)} \right] \right\}$$

To obtain the stress equations for a circle,
set $a=b$ in the stress equations for an ellipse.

Isotropic Stress Equations (Ellipse)

For an isotropic material:

$$\alpha = \beta = \xi = 1$$

Therefore, specific solution (from previous) becomes:

$$F = R \left\{ \frac{Pb}{2\xi \left(e^{\frac{\phi}{2}} - e^{-\frac{\phi}{2}} \right)} \left(-\frac{1}{a - e^{\frac{\phi}{2}} \xi b} \left(\frac{1}{2} (z_1 - w_1)^2 + y_1^2 \log(z_1 + w_1) \right) + \frac{1}{a - e^{-\frac{\phi}{2}} \xi b} \left(\frac{1}{2} (z_2 - w_2)^2 + y_2^2 \log(z_2 + w_2) \right) \right) \right\} + \frac{P\eta^2}{2\xi^2}$$

Isotropic Stress Equations (Ellipse)

$$F = \lim_{\phi \rightarrow 0} R \left\{ \frac{Pb}{\xi \left(e^{\frac{\phi}{2}} + e^{-\frac{\phi}{2}} \right)} \left[\begin{aligned} & - \frac{1}{a - e^{\frac{\phi}{2}} \xi b} \left(\frac{(z_1 - w_1)}{2} \left(ie^{\frac{\phi}{2}} \eta - \frac{z_1 ie^{\frac{\phi}{2}} \eta + e^{\phi} \xi^2 b^2}{w_1} \right) \right. \right. \\ & \quad \left. \left. + \frac{y_1^2}{2} \frac{ie^{\frac{\phi}{2}} \eta + \frac{z_1 ie^{\frac{\phi}{2}} \eta + e^{\phi} \xi^2 b^2}{w_1}}{z_1 + w_1} - e^{\phi} \xi^2 b^2 \log(z_1 + w_1) \right) \right. \\ & - \frac{e^{\frac{\phi}{2}} \xi b}{2 \left(a - e^{\frac{\phi}{2}} \xi b \right)^2} \left(\frac{(z_1 - w_1)^2}{2} + y_1^2 \log(z_1 + w_1) \right) \\ & + \frac{1}{a - e^{-\frac{\phi}{2}} \xi b} \left(\frac{(z_2 - w_2)}{2} \left(-ie^{-\frac{\phi}{2}} \eta + \frac{z_2 ie^{-\frac{\phi}{2}} \eta + e^{-\phi} \xi^2 b^2}{w_2} \right) \right. \\ & \quad \left. \left. + \frac{y_2^2}{2} \frac{-ie^{-\frac{\phi}{2}} \eta - \frac{z_2 ie^{-\frac{\phi}{2}} \eta + e^{-\phi} \xi^2 b^2}{w_2}}{z_2 + w_2} + e^{-\phi} \xi^2 b^2 \log(z_2 + w_2) \right) \right. \\ & - \frac{e^{-\frac{\phi}{2}} \xi b}{2 \left(a - e^{-\frac{\phi}{2}} \xi b \right)^2} \left(\frac{1}{2} (z_2 - w_2)^2 + y_2^2 \log(z_2 + w_2) \right) \end{aligned} \right] \right\} + \frac{P\eta^2}{2\xi^2}$$

Isotropic Stress Equations (Ellipse)

$$F = R \left\{ \frac{Pb}{2} \left[-\frac{1}{a-b} \left((z-w) \frac{(iyw - iyz - b^2)}{w} \right) - \frac{b}{2(a-b)^2} (z-w)^2 \right] \right. \\ \left. - \frac{a+b}{w} \left(iy + \frac{b^2}{z+w} \right) - b \log(z+w) + \frac{y^2}{b} \right\}$$

where:

$$w = \sqrt{z^2 - a^2 + b^2}$$

For simplification, set a=b to obtain stress equations for a circle:

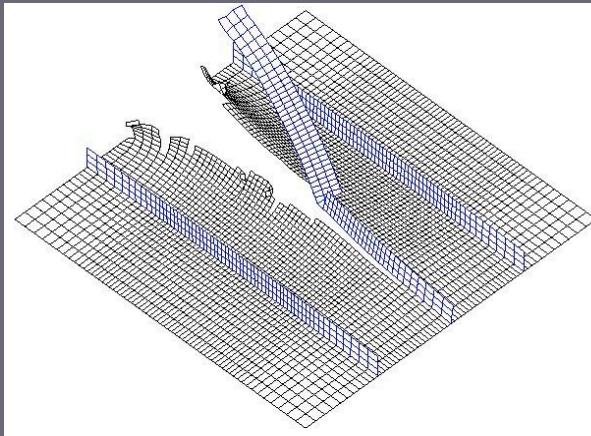
$$F = R \left\{ \frac{Pa}{2} \left(\frac{a^3}{z^2} - \frac{a^3}{2z^2} - \frac{2ay}{z} - \frac{a^3}{z^2} - a \log 2z + \frac{y^2}{a} \right) \right\}$$

In Polar Coordinates:

$$F = \frac{P}{4} \left(-\frac{a^4}{r^2} \cos 2\theta - 4a^2 \sin^2 \theta + 2r^2 \sin^2 \theta - 2a^2 \log r \right) \\ = \frac{P}{4} \left(r^2 - 2a^2 \log r - \frac{(r^2 - a^2)^2}{r^2} \cos 2\theta \right)$$

Extra: Fracture due to Crack-Propagation

- Fast Fracture of aircraft fuselage due to Crack Propagation
- Linear-Elastic Fracture Mechanics (LEFM)



Condition for Fast Fracture

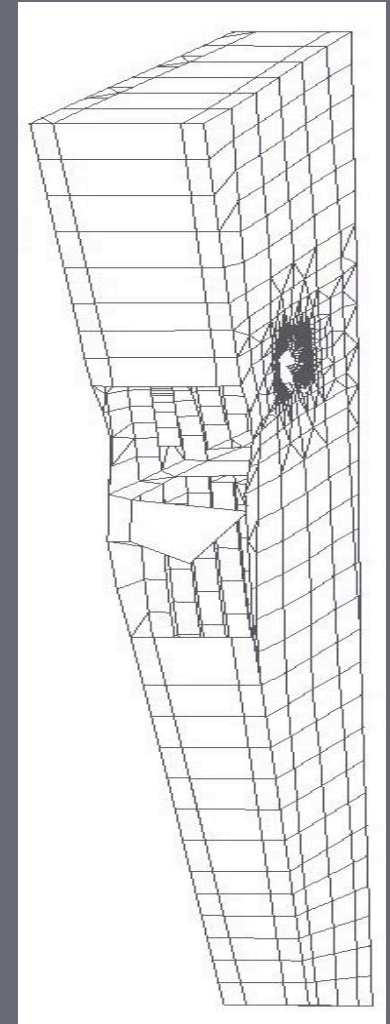
$$K_{1c} = Y\sigma\sqrt{\pi * a}$$

Plane
Strain:

$$a_c = \frac{1}{\pi} \left(\frac{K_{1c}}{Y\sigma} \right)^2$$

Plane
Stress:

$$r_Y = \frac{1}{2\pi} \left(\frac{K_c}{\sigma_Y} \right)^2$$



References

- ES 240 Lecture Notes provided by Professor Joost Vlassak
- “Theory of Plates and Shells” by Timoshenko and Woinowsky-Krieger
- “Elementary Engineering Fracture Mechanics” by David Broek
- “Engineering Materials 3: Materials Failure Analysis” by D.R.H. Jones